

Marco Rossi · Marco Sasso · Nathanael Connesson · Raman Singh
Adrian DeWald · David Backman · Paul Gloeckner *Editors*

Residual Stress, Thermomechanics & Infrared Imaging, Hybrid Techniques and Inverse Problems, Volume 8

Proceedings of the 2013 Annual Conference on Experimental
and Applied Mechanics



Conference Proceedings of the Society for Experimental Mechanics Series

Series Editor

Tom Proulx

Society for Experimental Mechanics, Inc.,

Bethel, CT, USA

For further volumes:

<http://www.springer.com/series/8922>

Marco Rossi • Marco Sasso • Nathanael Connesson • Raman Singh
Adrian DeWald • David Backman • Paul Gloeckner
Editors

Residual Stress, Thermomechanics & Infrared Imaging, Hybrid Techniques and Inverse Problems, Volume 8

Proceedings of the 2013 Annual Conference on Experimental
and Applied Mechanics

Editors

Marco Rossi
Università Politecnica delle Marche
Ancona, PA
Italy

Marco Sasso
Università Politecnica delle Marche
Ancona, PA
Italy

Nathanael Connesson
University of Joseph Fourier
Grenoble
France

Raman Singh
Oklahoma State University
Tulsa, OK
USA

Adrian DeWald
Hill Engineering, LLC
Rancho Cordova, CA
USA

David Backman
National Research Council Canada
Ottawa, ON
Canada

Paul Gloeckner
Cummins, Inc.
Columbus, IN
USA

ISSN 2191-5644
ISBN 978-3-319-00875-2
DOI 10.1007/978-3-319-00876-9
Springer Cham Heidelberg New York Dordrecht London

ISSN 2191-5652 (electronic)
ISBN 978-3-319-00876-9 (eBook)

Library of Congress Control Number: 2011922268

© The Society for Experimental Mechanics, Inc. 2014

This work is subject to copyright. All rights are reserved by the Publisher, whether the whole or part of the material is concerned, specifically the rights of translation, reprinting, reuse of illustrations, recitation, broadcasting, reproduction on microfilms or in any other physical way, and transmission or information storage and retrieval, electronic adaptation, computer software, or by similar or dissimilar methodology now known or hereafter developed. Exempted from this legal reservation are brief excerpts in connection with reviews or scholarly analysis or material supplied specifically for the purpose of being entered and executed on a computer system, for exclusive use by the purchaser of the work. Duplication of this publication or parts thereof is permitted only under the provisions of the Copyright Law of the Publisher's location, in its current version, and permission for use must always be obtained from Springer. Permissions for use may be obtained through RightsLink at the Copyright Clearance Center. Violations are liable to prosecution under the respective Copyright Law.

The use of general descriptive names, registered names, trademarks, service marks, etc. in this publication does not imply, even in the absence of a specific statement, that such names are exempt from the relevant protective laws and regulations and therefore free for general use.

While the advice and information in this book are believed to be true and accurate at the date of publication, neither the authors nor the editors nor the publisher can accept any legal responsibility for any errors or omissions that may be made. The publisher makes no warranty, express or implied, with respect to the material contained herein.

Printed on acid-free paper

Springer is part of Springer Science+Business Media (www.springer.com)

Preface

Residual Stress, Thermomechanics & Infrared Imaging, Hybrid Techniques and Inverse Problems, Volume 8: Proceedings of the 2013 Annual Conference on Experimental and Applied Mechanics represents one of eight volumes of technical papers presented at the SEM 2013 Annual Conference & Exposition on Experimental and Applied Mechanics organized by the Society for Experimental Mechanics and held in Lombard, IL, June 3–5, 2013. The complete Proceedings also includes volumes on: *Dynamic Behavior of Materials; Challenges in Mechanics of Time-Dependent Materials and Processes in Conventional and Multifunctional Materials; Advancement of Optical Methods in Experimental Mechanics; Mechanics of Biological Systems and Materials; MEMS and Nanotechnology; Experimental Mechanics of Composite, Hybrid, and Multifunctional Materials; Fracture and Fatigue.*

Each collection presents early findings from experimental and computational investigations on an important area within Experimental Mechanics, Residual Stress, Thermomechanics & Infrared Imaging, Hybrid Techniques and Inverse Problems being three of these areas.

Residual stresses have a great deal of importance in engineering systems and design. The hidden character of residual stresses often causes them to be underrated or overlooked. However, they profoundly influence structural design and substantially affect strength, fatigue life, and dimensional stability. Since residual stresses are induced during almost all materials processing procedures, for example, welding/joining, casting, thermal conditioning, and forming, they must be taken seriously and included in practical applications.

In recent years, the application of infrared imaging techniques to the mechanics of materials and structures has grown considerably. The expansion is marked by the increased spatial and temporal resolution of the infrared detectors, faster processing times, and much greater temperature resolution. The improved sensitivity and more reliable temperature calibrations of the devices have meant that more accurate data can be obtained than were previously available.

Advances in inverse identification have been coupled with optical methods that provide surface deformation measurements and volumetric measurements of materials. In particular, inverse methodology was developed to more fully use the dense spatial data provided by optical methods to identify mechanical constitutive parameters of materials. Since its beginnings during the 1980s, creativity in inverse methods has led to applications in a wide range of materials, with many different constitutive relationships, across material heterogeneous interfaces. Complex test fixtures have been implemented to produce the necessary strain fields for identification. Force reconstruction has been developed for high strain rate testing. As developments in optical methods improve for both very large and very small length scales, applications of inverse identification expand to include geological and atomistic events.

Ancona, PA, Italy
Ancona, PA, Italy
Grenoble, France
Tulsa, OK, USA
Rancho Cordova, CA, USA
Ottawa, ON, Canada
Columbus, IN, USA

Marco Rossi
Marco Sasso
Nathanael Connesson
Raman Singh
Adrian DeWald
David Backman
Paul Gloeckner

Contents

1	Analysis of Thrust Production in Small Synthetic Flapping Wings	1
	Kelvin Chang, Jason Rue, Peter Ifju, Raphael Haftka, Tony Schmitz, Chris Tyler, Anirban Chaudhuri, and Vasishta Ganguly	
2	Coarse-Resolution Cone-Beam Scanning of Logs Using Eulerian CT Reconstruction. Part I: Discretization and Algorithm	9
	Yuntao An and Gary S. Schajer	
3	Coarse-Resolution Cone-Beam Scanning of Logs Using Eulerian CT Reconstruction. Part II: Hardware Design and Demonstration	21
	Yuntao An and Gary S. Schajer	
4	Crack Nucleation Threshold Under Fretting Loading by a Thermal Method	31
	B. Berthel and S. Fouvry	
5	Crack Growth Study of Fibre Metal Laminates Using Thermoelastic Stress Analysis	39
	Rachel A. Tomlinson, Khurram Amjad, and Gonzalo Genovart Urria	
6	Crack Detection in Large Welded Components Under Fatigue Using TSA	47
	Andrew Robinson, Janice Dulieu-Barton, Simon Quinn, and David Howarth	
7	Hybrid Thermoelastic Analysis of an Unsymmetrically-Loaded Structure Containing an Arbitrarily-Shaped Cutout	51
	W.A. Samad and R.E. Rowlands	
8	Quantitative Thermographic Characterization of Composites	59
	Steven M. Shepard	
9	Thermal Deformation of Micro-structure Diffuser Plate in LED Backlight Unit	67
	Chun-Cheng Chen, Chi-Hui Chien, Bo-Syun Chen, Ting-Hsuan Su, Jyun-Jie Li, Jia-Wei Che, Jie-Lin Yu, and Zhi-Yan Wang	
10	Polariscopy Measurement of Residual Stress in Thin Silicon Wafers	79
	K. Skenes, R.G.R. Prasath, and S. Danyluk	
11	Curvature from Residual Stress in Rapidly Quenched Amorphous Metals Using Abrasive Layer Removal	87
	Balaji Jayakumar and Jay C. Hanan	
12	Slitting Method Measurement of Residual Stress Profiles, Including Stress Discontinuities, in Layered Specimens	93
	Michael B. Prime and David L. Crane	
13	Repeatability of the Contour Method for Residual Stress Measurement	103
	Michael R. Hill and Mitchell D. Olson	

14	Repeatability of Incremental Hole Drilling and Slitting Method Residual Stress Measurements	113
	Adrian T. DeWald and Michael R. Hill	
15	Drilling Speed Effects on Accuracy of HD Residual Stress Measurements	119
	C. Barile, C. Casavola, G. Pappalettera, C. Pappalettere, and F. Tursi	
16	Ultrasonic Nonlinear Guided Waves and Applications to Structural Health Monitoring	127
	Claudio Nucera and Francesco Lanza di Scalea	
17	Defect Detection in Composite Structures Using Lock-In Infrared Thermography	135
	Arun Manohar and Francesco Lanza di Scalea	
18	Digital Image Correlation for Identification of Vibration Modes	143
	Olli Puustinen	
19	Application of VFM for the Simultaneous Identification of Visco-pseudo-hyper Elastic Constants of Rubbers	153
	G. Chiappini, M. Sasso, M. Rossi, and D. Amodio	
20	Anisotropy Evaluation of Paperboard With Virtual Fields Method	163
	J.M. Considine, F. Pierron, K.T. Turner, and D.W. Vahey	
21	Investigation of Fatigue Properties of Titanium Alloy Applying Variational Infrared Thermography	171
	P.J. Hou, X.L. Guo, and J.L. Fan	
22	Monitoring Materials Under Impact with Infrared Thermography	177
	Carosena Meola and Giovanni Maria Carlomagno	
23	Damage Induced Evolution of the Thermal Diffusivity of SiC/SiC Composite	187
	Jalal El Yagoubi, Jacques Lamon, Jean-Christophe Batsale, and Marion Le Flem	
24	Identification of Welding Heat Sources from Infrared Temperature Measurements	199
	B. Beaubier, K. Lavernhe-Taillard, R. Billardon, C. Boucq, P. Laloue, and B. Darciaux	
25	Nondestructive Evaluation of Fiber Reinforced Polymers with Lockin Thermography	207
	Carosena Meola, Veronica Grasso, Cinzia Toscano, and Giovanni Maria Carlomagno	
26	Realistic 3D FE Modelling of Peening Residual Stresses of Strain-Rate Sensitive Materials with Oblique Incident Angles	215
	F. Yang, Z. Chen, S.A. Meguid, and M. Guagliano	
27	Water Cavitation Peening by Ultrasonic Vibration for Improvement of Fatigue Strength of Stainless Steel Sheet	221
	Tomohiro Sasaki, Shun Hasegawa, Masayuki Nakagawa, and Sanichiro Yoshida	
28	Effect of Residual Stress on Spallation of NiCrBSi Coating	229
	Chen-Wu WU	
29	Multiaxial Fatigue Resistance of Shot Peened High-Strength Aluminium Alloys	235
	M. Benedetti, V. Fontanari, D. Bergamini, M. Bandini, and D. Taylor	
30	Quantifying Residual Strains in Specimens Prepared by Additive Layer Manufacturing	245
	A.N. Okioga, R.J. Greene, and R.A. Tomlinson	
31	Parameter Determination of Anisotropic Yield Criterion	253
	Jin-Hwan Kim, Frédéric Barlat, and Fabrice Pierron	
32	Performance Assessment of Inverse Methods in Large Strain Plasticity	259
	Marco Rossi, Marco Sasso, Gianluca Chiappini, Dario Amodio, and Fabrice Pierron	

33	Optical: Numerical Determination of the Flow Curves of Anisotropic Steels and Failure Prediction	267
	G. Mirone	
34	Advanced Biaxial Cruciform Testing at the NIST Center for Automotive Lightweighting	277
	Mark A. Iadicola, Adam A. Creuziger, and Tim Foecke	
35	An Enhanced Plasticity Model for Material Characterization at Large Strain	287
	L. Cortese, G.B. Broggiato, T. Coppola, and F. Campanelli	
36	Residual Stress and Phase Transformation Map for Impact Fatigued Zirconia	295
	M. Allahkarami and J.C. Hanan	
37	Characterization of Silicon Photovoltaic Wafers Using Infrared Photoelasticity	303
	T.-W. Lin, G.P. Horn, and H.T. Johnson	
38	Acoustic Emission Analysis of Aluminum Specimen Subjected to Laser Annealing	309
	C. Barile, C. Casavola, G. Pappalettera, and C. Pappalettere	
39	Optical Analysis of Weld-Induced Residual Stress by Electronic Speckle-Pattern Interferometry	317
	Sean Craft, Saugat Ghimire, Bishwas Ghimire, T. Sasaki, and Sanichiro Yoshida	
40	Shape Is Not Enough to Test Hypotheses for Morphogenesis	325
	Victor D. Varner, Gang Xu, and Larry A. Taber	
41	FBG Based In Situ Characterization of Residual Strains in FDM Process	333
	Antreas Kantaros and Dimitris Karalekas	
42	Polynomial Fitting Techniques for IRT Inspection	339
	Shanglei Li, Anish Poudel, and Tsuchin Philip Chu	
43	In-Flight Infrared Thermography for Studies of Aircraft Cabin Ventilation	347
	T. Dehne, J. Bosbach, and A. Heider	
44	Study of Elastomeric Membranes for Vibration Dampers in Non-stationary Conditions	355
	M. Sasso, G. Chiappini, M. Rossi, and E. Mancini	
45	Structural Dynamic Modification to Predict Modal Parameters of Multiple Beams	365
	Naim Khader	

Chapter 1

Analysis of Thrust Production in Small Synthetic Flapping Wings

Kelvin Chang, Jason Rue, Peter Ifju, Raphael Haftka, Tony Schmitz, Chris Tyler, Anirban Chaudhuri, and Vasishtha Ganguly

Abstract For flapping micro air vehicles, geometrical parameters such as size, aspect ratio as well as structural topology can affect thrust production in hover mode. Synthetic wings similar in size to that of a humming bird's were manufactured with the hope of understanding these affects. The experimental method for measuring thrust and the manufacturing process used to make the wings have seen improvement from previous work such that there is less scatter and uncertainty; this allows for smaller variations in thrust to be detected. With confidence in the fabrication and testing procedure, an optimization problem was attempted where three design parameters were chosen as variables and the objective was to maximize thrust. These efforts were coupled with noncontact imaging techniques like digital image correlation and laser doppler velocimetry to help extract the characteristics that are consistent with wings that produce considerable thrust. The results of these tests will help to obtain the relationships between the consciously selected geometric parameters and the thrust produced. It was found that by machining the synthetic wings from acetal resin sheet and pairing that skeleton with a carbon fiber rod less variation was present. This wing construction was found to have a quick production time, making an experimental optimization feasible.

Keywords MAV • Flapping • Thrust • DIC • LDV

1.1 Introduction

Small flapping wing micro air vehicles (MAV) are continuously filling new roles and are presenting extraordinary functions. Theoretically, the possibilities for flapping wing MAVs stretch much further than previous aerial vehicles since they may have accumulative advantages of both rotary wing and fixed wing designs. These include efficiencies, noise level, and maneuverability although research is still being conducted to learn the physics behind flapping flight. In a study comparing flapping wing MAVs and conventional fixed wing MAVs, Hu et al. found definitive benefits in flapping flight below an advance ratio of 1.0 (forward flight speed to wingtip velocity) [1]. For this study, an unsteady state regime is investigated with advance ratio closer to 0 (hovering flight) with focus on the thrust produced by wings possessing wingspans no longer than 230 mm (~9 in.) at the highest aspect ratio. The purpose of this experiment was to take an enhanced manufacturing process along with digital image correlation, vibrational analysis, thrust testing, and optimization techniques to correlate certain variables to thrust production [2]. With in-site into the parameters that correlate well with thrust, new knowledge of desirable design features and reinforcement can be developed.

K. Chang (✉) • J. Rue • P. Ifju • R. Haftka • A. Chaudhuri
Mechanical and Aerospace Engineering Department, University of Florida, Florida, USA
e-mail: kc3635@ufl.edu

T. Schmitz • C. Tyler • V. Ganguly
Mechanical and Aerospace Engineering Department, University of North Carolina in Charlotte, Florida, USA

1.2 Advances in Manufacturing Process

The wings take advantage of passive wing deformation, much like hummingbirds or insects, as described by Wu [3]. The wing compliance and leading edge stiffness have positive aerodynamic effects and, while using passive deformation, can simplify kinematic actuation while reducing parts and weight. Having wings that are repeatable and robust are essential to any further developments in this area. The first step to creating a high-fidelity experimental process was to improve a commonly used method for wing assembly, as Nguyen performed [4]. A hand lay-up technique where carbon fiber strips are delicately laid on a flat plate in a certain pattern and cured under pressure in an oven is typical but holds many uncertainties and concerns. Details of how the hand lay-up method can lead to frequent problems and inconsistencies are described by Rue [5] along with an iteration using a CNC milled Teflon mold.

Figure 1.1 below shows an apparent difference, even visually, amongst a wing created with the original hand lay-up process and one with the CNC milled Teflon mold.

Several mold options had progressed to fix arising problems such as uneven pressure distributions, warping, or thermal expansion, although, ultimately, cost and precision needs drove the study to the current method. As seen in Fig. 1.2, a CNC milled Delrin (Acetal Resin) [6] frame is constructed and a pre-cured commercially available carbon fiber circular rod is

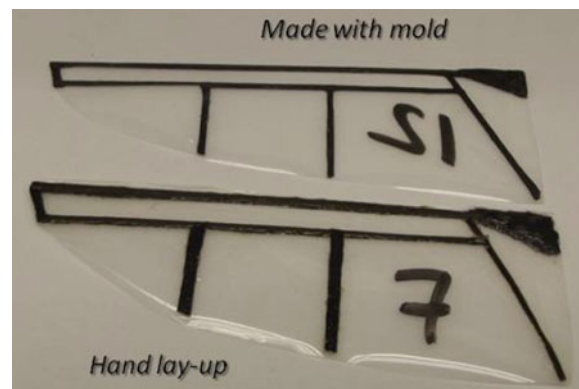


Fig. 1.1 Comparison of the original hand lay-up process with the CNC milled carbon fiber wing showing a clear distinction between the two in terms of precision and repeatability. (These wings are 75 mm in length)

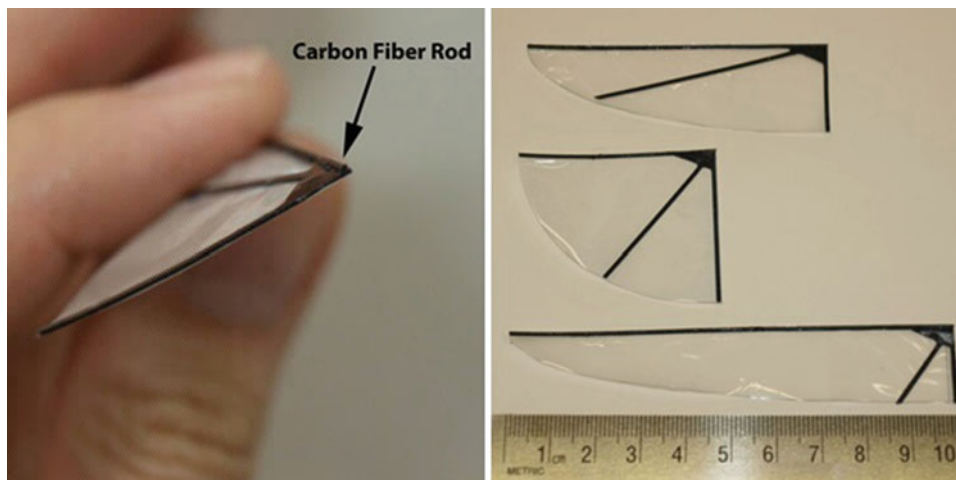


Fig. 1.2 Pictured on the *left* is a CNC milled plastic wing with a circular carbon fiber rod attached along the length of the leading edge. On the *right* is an image that depicts three of the six different aspect ratios tested with each wing possessing various batten angle degrees and leading edge rod lengths

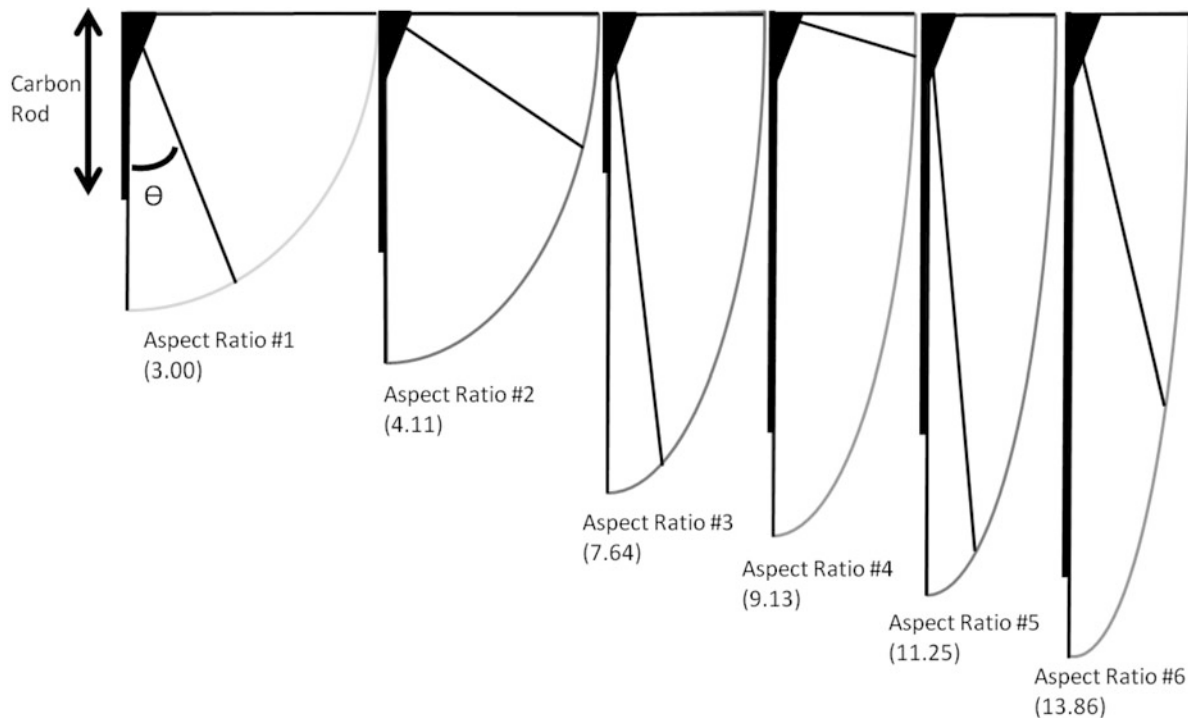


Fig. 1.3 Three chosen variables that could be manufactured and tested within the first round of optimization. Each of the six aspect ratios has the same quarter-elliptical area and three main components: a Delrin plastic frame, a 0.02 in. circular diameter carbon fiber rod, and a nylon based membrane called Capran. Each has a leading edge, a root batten, a triangular attachment point, and a batten elongating at a certain angle, θ , from the leading edge

attached in a trough on the leading edge. This process does not require any curing cycle for the carbon fiber and adds the precision and repeatability capability of a CNC to the wing. The weight of the wings remains compare well with the previous carbon fiber wings as well, ranging from approximately 0.1 to 0.25 g per wing. To keep a solid bond between the frame and rod, a rubber toughened cyanoacrylate glue was applied. Identical to the all carbon construct wing, the completed frame is glued down to a nylon based, 14 μm thick, membrane created by Honeywell, called Capran [7]. The wing area has an affect on the thrust output, so to maximize repeatability and to further development; thin marks are printed onto the Capran as cutting guides. An ink jet printer is utilized so that heat does not affect the material properties of the membrane.

Once the manufacturing was acceptable in terms of how repeatable the thrust measurements became, three variables were chosen for the initial optimization period (Fig. 1.3). These consist of varying the carbon fiber rod length on the leading edge, the angle of the one batten, and the aspect ratio. The aerodynamic definition for aspect ratio ($\frac{b^2}{S} \equiv \frac{\text{span}^2}{\text{wing area}}$) was used for this term. These three variables are the bases of the optimization approach used, allowing for a consistent manufacturing process that is characterized by little minimal human error.

1.3 Vibration Analysis

Given the various actuation frequencies that the wings are exposed to, an investigation of the mode shapes and natural frequencies of the wings were put in place to assist in developing a better understanding of the wing's dynamics. The added benefit of this experiment includes the possibility of finding a correlation between these vibration parameters and the thrust produced.

A Polytec scanning vibrometer is used in the experimental setup for testing the wings which includes a PSV-I-400 scanning head, a PSV-400 junction box, and a OFV-500 controller module. These components work alongside a Low Dynamic Stiffness (LDS) V201 permanent magnet shaker that possesses a useful frequency range between 5 Hz and 12 kHz and is mounted on a trunnion for stability [8]. The shaker assembly was placed on a Newport RS1000 Research grade optical table Fig. 1.4.

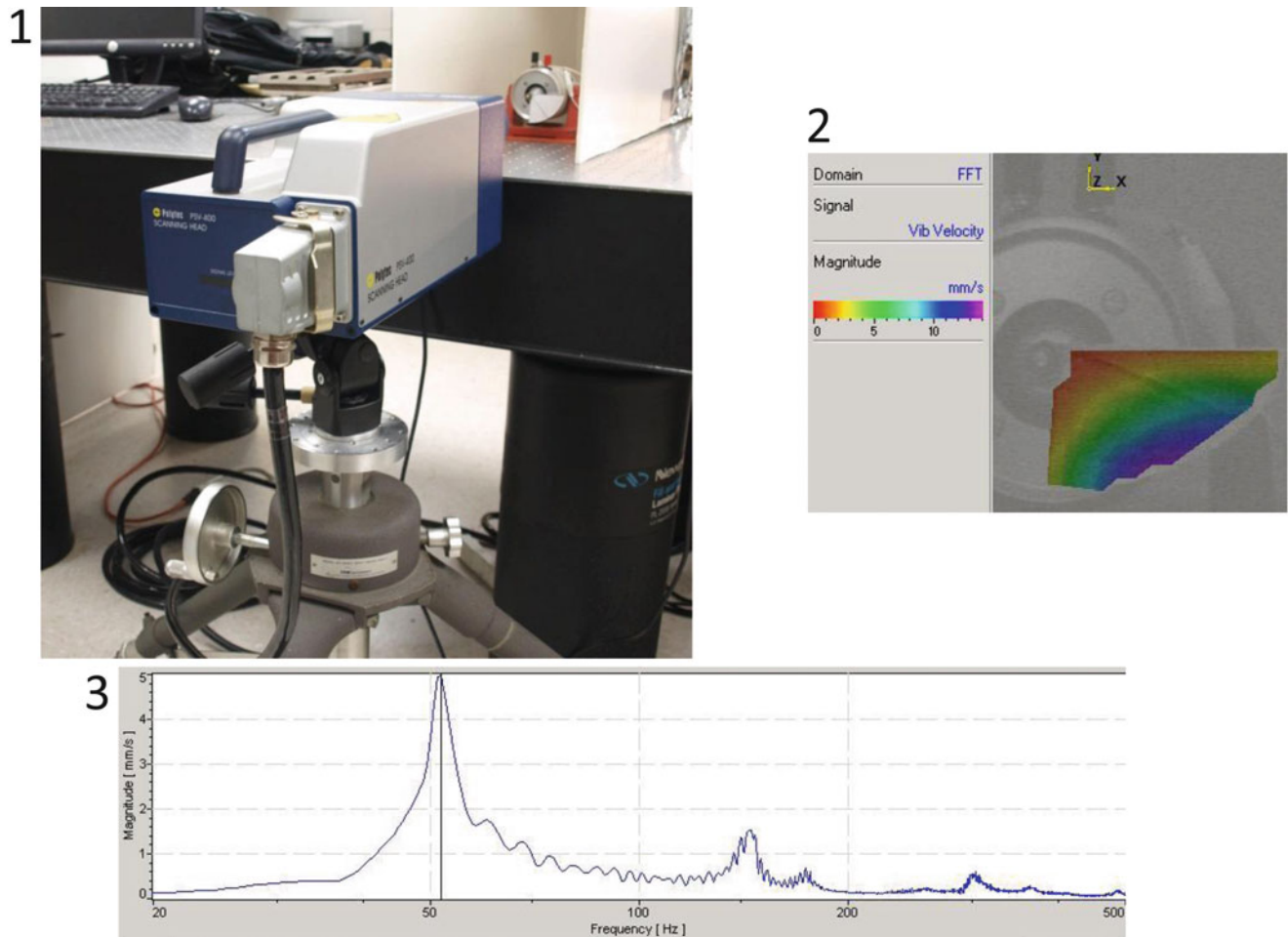


Fig. 1.4 (1) The Polytec scanning head, mounted on a tripod, is pictured with a shaker that linearly actuates the attached wing. (2) A typical instantaneous velocity color plot. (3) FFT graph showing the first three natural frequencies

Each wing is attached to the shaker assembly with a rigid bolt system and is lightly painted to increase the opacity of the surface. Ideally, the exposed surface would be physically flat and matte in finish. The experiment was performed in a dim external light setting to reduce any possible noise. Along with that, an enclosure was developed to shield it from excess wind created by the room's air conditioner unit. For each study, a grid of sampling points was populated on the wing. The shaker was commanded to sweep between 1 Hz and 500 Hz in a 1.6 s time period. Each point over the wing was tested five times to conclude in an averaged result. Data collected in this manner automatically generates an FFT graph, showing the modal frequencies for each wing (Fig. 1.4).

1.4 Preliminary Power Investigation

The controller software used with the flapping mechanism (Fig. 1.5) serves to obtain a commanded motor velocity for the wings by taking information from an encoder mounted on the back of the motor and devising a response in the form of an output current. For each of the wings, it becomes apparent that unequal quantities of power are being consumed by each trial, including discrepancies between different wing designs. Preliminary studies suggest that power consumed by a wing flapping at 10 hz, in a 5 s window, can vary by as much as 45 %, depending on controller response.

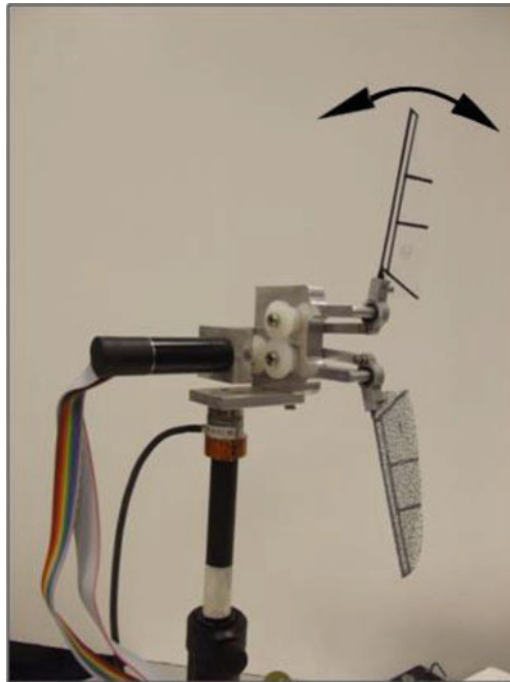


Fig. 1.5 A figure containing the flapping mechanism which entails an aluminum stand, an ATI Nano17 6 axis sensor (with *black cord* extending to the left), Maxon motor, and aluminum mechanism with nylon gearing system. This coupled with dual LabVIEW programs allows for the generated thrust and lift to be read and processed through MATLAB

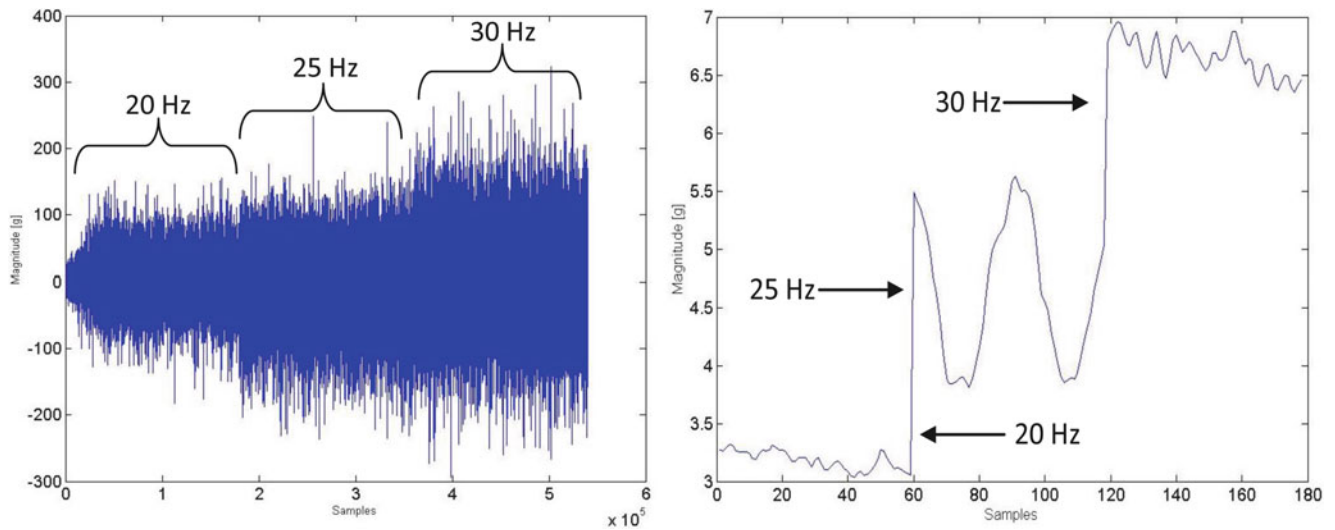


Fig. 1.6 Shows a view of two thrust data output examples, unfiltered on the *left* and low pass filtered on the *right*. Calculations done on the same wing give the same averaged thrust values for each frequency, though clearer distinctions between the three different frequency regimes are observed. Note that these are not of the same wing

1.5 Thrust Acquisition

Thrust measurements acquired within this experiment are gathered through two LabVIEW programs. Each wing outputs raw data at a sampling rate of 30,000 samples per second for several flapping frequencies—controlled by the Maxon motor and LabVIEW—for both thrust and lift. The raw data is then averaged over the specific run time for a single averaged thrust/lift value for each frequency. The flapping mechanism is pictured in Fig. 1.5.

The output signal from the load cell possesses a considerable amount of noise; making it very difficult to interpret. Therefore, a filter to extract a more manageable signal was required (Fig. 1.6). A point by point low pass Butterworth filter

was used real time in the LabVIEW software code to process the noisy signal to present the low frequency thrust offset data that is desired for each frequency. This filter provides the operator of the experiment real time performance that is easy to interpret, serving as a check to assure consistency and create a warning sign for possible delamination or damage to the wing between trials.

1.6 Digital Image Correlation

Based on previous studies, the leading edge stiffness of a wing has been proven to correlate with thrust production. Unique to this study, only a percentage of the leading edge length is fortified by the stiffness of a circular cross section carbon fiber rod. For each of the wings tested for thrust, digital image correlation (DIC) was utilized to extract deflections resulting from standardized weights (0.3 g, 0.5 g, 1.0 g) attached to the wing by a Kevlar string. The intent of this is find correlation between these deflections and the thrust output.

For the setup, each wing is misted with black paint and thin Kevlar strings are attached at the root and leading edge with thin cyanoacrylate glue, allowing for the application of the loads. The experimental setup includes two Point Grey Research Flea2 cameras that are aimed at the matte side of the Capran wing skin. Correlated Solution's VIC 2010 picture capturing software is used to take stills that are processed in VIC-3d. Two pictures at every load state are taken and the deflection results from these are averaged to reduce error (Fig. 1.7).

The resulting tip deflection due to a 0.5 g weight plotted against the percentage of the plastic leading edge that is fortified provides insight into the behavior of the wings. The bubble plot, Fig. 1.8, follows an exponential curvature with an

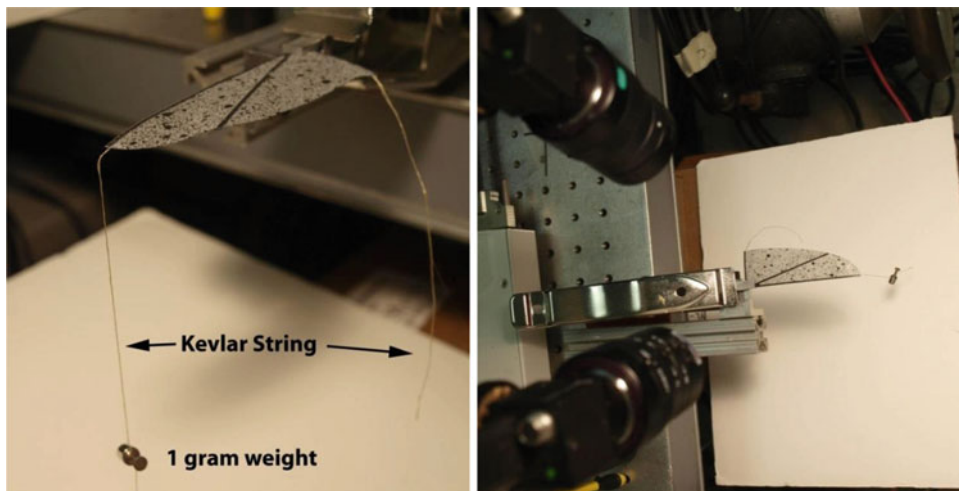


Fig. 1.7 On the *left*, a wing clamped in place and tested using DIC. On the *right* is an image of the DIC setup used to test for deflections. Speckled with enamel paint, the wing is subjected to a 1 g load at the tip of its leading edge via a Kevlar string

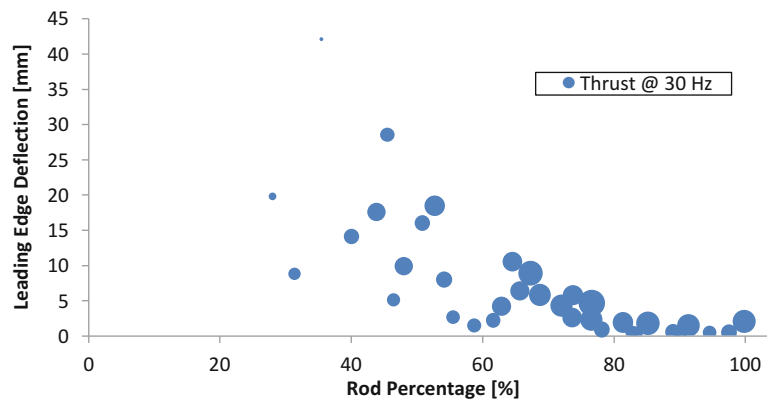


Fig. 1.8 A bubble plot presents the deflection data with a 0.5 g load obtained for different carbon rod percentages that occupy the plastic leading edge. The sizes of the bubbles illustrate the amount of thrust obtained for the wings

asymptote at zero deflection, indicating little deflection for wings with close to 100 % fortification. This expected result brings validation to the procedure used to gather the deflection. Another interesting behavior of this plot is how some of the data points have drifted to the left of the densely populated curve. This suggests that variables not considered in this plot have reduced the deflection. There are reasons to believe that the batten angle variable played a role in reducing the deflection of some of the points described in the plot.

1.7 Optimization

There is a multitude of variables that lend to the resulting thrust produced by a specific wing design. This complexity has encouraged efforts to optimize in a three variable space, seen in Fig. 1.9, focusing on variables that have proven to be strongly correlated with thrust output. The advantage of this is in the ease of populating the design space in a reasonable amount of time, and that the results of this smaller study could lend to larger ones where previously constrained variables could be allowed to vary. A technique called Efficient Global Optimization (EGO) is used to determine a set of wing designs that are good candidates for producing more thrust than the best designs found in previous tests. In batches of 20 new designs, the wings were tested for thrust production and a host of other parameters explained previously. EGO is able to conjure designs by considering previous tests. Therefore, the optimization is guided, pivoting off of results obtained from previous batches.

Table 1.1 explains the initial design space for the first 20 wings. After extensive testing, the space was altered slightly to exclude unreasonable regions for wings 21–40. For example, the leading edge stiffness percentage range was reduced to 40–100 % since. Even though the prior range was feasible to manufacture, the excess deformation and poor results in flapping led to concerns. Also, the angle of the batten was changed to a minimum of 10° to address structural support along the trailing edge.

Fig. 1.9 Three dimensional design space with equally spaced specimens. Variables include aspect ratio, the percentage of the leading edge that the carbon fiber rod extends (commercially available), and the angle of the one batten, measured from the leading edge. For a detailed illustration, refer to Fig. 1.3

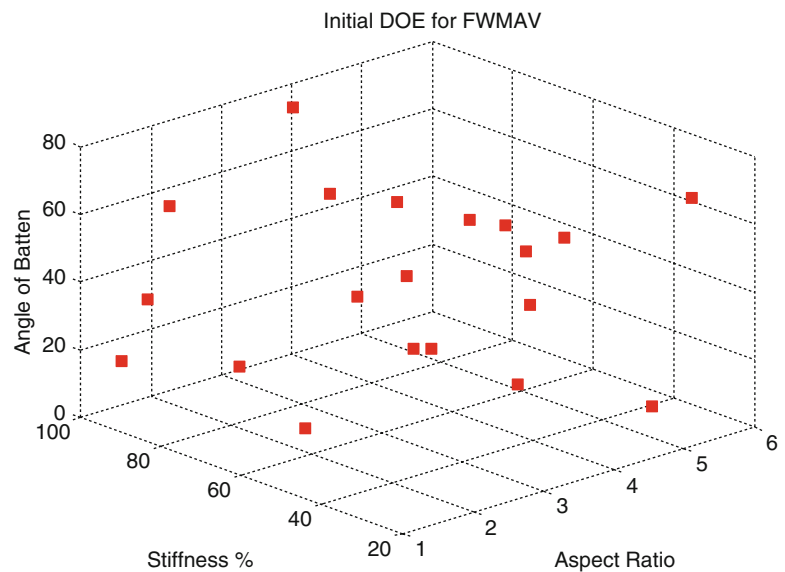


Table 1.1 The table below details the three variables that dictate the design of each wing in the single objective, multi-variable optimization

Design space		
Variable	Range	Description
Aspect ratio	[0.55, 6.45]	Contains six options of aspect ratio, rounded to the nearest one
Stiffness percentage	[25, 100]	Percentage of the leading edge that a commercially available carbon fiber rod extends
Angle of batten	[0.55, 80.45]	Angle of the one batten, measured from the leading edge

1.8 Future Work

The optimization process is continuing and will hopefully create new knowledge for the design of wing reinforcement. This study will continue to explore other variables that correlate with thrust production, expanding the optimization space in areas such as larger aspect ratios. Along with that, future advances in the flapping mechanism are planned, hoping to increase the reliability, control, and function of the device. One feature in particular that is of interest is the capability of commanding wing position in the axis perpendicular to the flapping axis which is much in the same capability of natural fliers. In regard to the optimization, the guide algorithm suggests higher thrust production in wings with aspect ratios beyond those chosen for our study, so investigation of that is expected. Ultimately, this study hopes to create wings for usage on flapping micro air vehicles.

References

1. Hu H, Kumar AG, Abate G, Albertani R (2010) An experimental investigation on the aerodynamic performances of flexible membrane wings in flapping flight. *Aerosp Sci Technol* 14:575–586
2. Ifju P, Schmitz T, Haftka R (2010) Expanding the design space of synthetic flexible flapping wings by advancing fabrication and optimization methodologies. University of Florida, AFOSR/RSA Proposal
3. Wu P (2010) Experimental characterization, design, analysis and optimization of flexible flapping wings for micro air vehicles. Dissertation, University of Florida, Gainesville
4. Nguyen Q, Park H, Goo N, Byun D (2010) Characteristics of a Beetle's free flight and a flapping-wing system that mimics beetle flight. *J Bionic Eng* 7(1):77–86
5. Rue J, Chang K (2012) Fabrication and analysis of small flapping wings. In: 2012 annual conference on experimental and applied mechanics, Costa Mesa
6. McMaster-Carr (2012) More about plastics [Online]. Available <http://www.mcmaster.com/#8574kac/=Inff8t>. Accessed 26 Feb 2013
7. Honeywell International Inc (2010) CAPRAN 1200 Matte [Online]. Available http://www51.honeywell.com/sm/capran/common/documents/PP_Capran_1200_matte_Specification_sheet.pdf. Accessed 26 Feb 2013
8. Bruel & Kjaer (2012) Bruel & Kjaer LDS v201 product data [Online]. Available (by request). Accessed Feb 2013

Chapter 2

Coarse-Resolution Cone-Beam Scanning of Logs Using Eulerian CT Reconstruction. Part I: Discretization and Algorithm

Yuntao An and Gary S. Schajer

Abstract Computed Tomography (CT) is extensively used as a medical diagnostic tool, and increasingly for scientific and industrial research. In the wood industry, there is a growing interest in using the CT technique to assess the quality of logs entering a sawmill. Internal features of interest include knots, heartwood/sapwood boundary, rot and splits. Most commercially available CT scanning systems are modeled on medical designs and provide high spatial and density resolution. However, they are very complex and delicate devices and their cost is correspondingly high. So far, there is no commercially available CT scanner that can meet the extreme scanning speed requirement, moderate affordability and severe working environment in a sawmill. To address these challenges for using CT technology for industrial log scanning, a novel coarse-resolution cone-beam CT scanning system has been developed. To accommodate the modestly accurate log transport systems in sawmills and hence address the substantial associated lateral motions, an Eulerian approach is taken whereby the CT reconstruction is based on the moving log rather than on the fixed space traversed by the log. This paper, the first of a two-part report, describes a novel cone beam scanning concept, geometry-based coarse-resolution log models, customized CT data processing, normalization and efficient cone beam reconstruction algorithms. The second part of this report will describe the construction details and practical performance of a prototype device.

Keywords Log sorting • Log scanning • Cone-beam CT • Coarse-resolution CT • Log feature identification

2.1 Introduction

Wood is a highly variable natural material that requires an individual decision for each wood piece to identify the most advantageous processing method. In this way the most appropriate and highest value products can be produced from the available raw material. At present, log inspection is based on visual observation of surface defects and optical measurement of external features [1]. The logs are then cut according to their observed characteristics. However, many quality-controlling features are not visible on the surface, causing the resulting cutting to be far from optimal. Studies indicate that only half of inspected logs are classified correctly by a human inspector [2]. Consequently, many logs are placed at the wrong breakdown position, dramatically reducing the amount of high-value products obtained [3, 4]. It is estimated that the value of sawn timber could increase by 7–15 % if the internal defects in logs were accurately known [5, 6]. This is a massive value increase and urgently points to the need for an effective log scanning tool.

Computed Tomography (CT) is a powerful technique to create 2-dimensional cross-sectional views of an object from multiple 1-dimensional X-ray measurements called “projections”. These 2-D views reveal the internal features within the object. Traditionally, CT has mainly been used as a medical diagnostic tool, but now is increasingly applied in industry. There is a growing interest in using the CT technique to assess log quality in sawmills, both on the measurement side [6, 7] and on the data analysis side [8–11]. CT is a powerful tool to identify quality-defining features such as knots, heartwood/sapwood extent, rot and splits, and it provides rich information to guide subsequent manufacturing of high-value products.

CT scanning systems applied to log-scanning applications are modeled on medical designs and have very high spatial and density resolution. Consequently, the equipment is complex, requires high-precision motions of sensors and

Y. An • G.S. Schajer (✉)

Department of Mechanical Engineering, University of British Columbia, Vancouver, BC, Canada

e-mail: schajer@mech.ubc.ca

specimen, and is very costly. In addition, the associated CT reconstruction is very computationally intensive. It requires massive data collection, processing and data analysis to identify features of interest within the CT images. These requirements do not fit well within a sawmill environment where the needs are for straightforward operation, tolerance of inaccurate motions, and moderate affordability. The logs must be scanned, analyzed and have sawing decisions made in real-time, often just 5–10 s per log. To address the challenges in applying CT technology to log scanning, a novel coarse-resolution cone-beam CT system has been designed and prototyped here. The proposed system uses straightforward and modest cost equipment, and uses advance (“a-priori”) knowledge of the specific geometry of saw logs to reduce dramatically the quantity and quality of data required, the required precision of the relative motion of the X-ray sensors and measured logs, and the scale of the CT image reconstruction computation. To accommodate the relative motion challenge, an Eulerian approach is taken whereby the CT reconstruction is based on the moving log rather than on the fixed space traversed by it. This paper is the first of a two-part series. It introduces the proposed cone-beam log scanning concept, the geometric log models, the associated algorithms and computation procedures.

2.2 Conventional CT Versus Cone-Beam Coarse-Resolution CT

Conventional CT scanning functions by making X-ray measurements from multiple directions and mathematically combining them to generate a 3-dimensional model of the scanned object [12]. Figure 2.1a, b respectively show a typical medical-style third generation fan-beam CT scanner and a typical cone-beam CT scanner. For both scanning geometries, the object of the measurement, commonly a patient in a hospital but here a log, stays at the center and an X-ray source and a line or area detector rotates around the outside to gather the multi-directional series of radiographs required for the CT reconstruction. These designs both require that the X-ray source and detector accurately rotate around the measured object so as to maintain accurate spatial registration among the radiographs measured from the various directions. The single-slice CT arrangement using a line-detector shown in Fig. 2.1a has to scan each cross-section individually and thus operates relatively slowly. It also uses only a small area-fraction of the emitted X-rays, and so forces the use of a much higher X-ray flux than required solely for the detectors. Use of an area detector such as in Fig. 2.1b with a cone-beam X-ray source allows a large volume of an object to be measured simultaneously while gaining data from a much greater fraction of the available X-rays. However, such systems are complex both in construction and in mathematical processing of the X-ray measurement. In addition, commercial cone-beam detectors rarely are larger than 12" (30 cm) across [12], and thus are not sufficient in size for saw-log scanning. The low end of the range for saw-logs is 10–18" (25–45 cm), so allowing for the cone angle and some free space at the edges, a minimum detector size of 24" (60 cm) across is required.

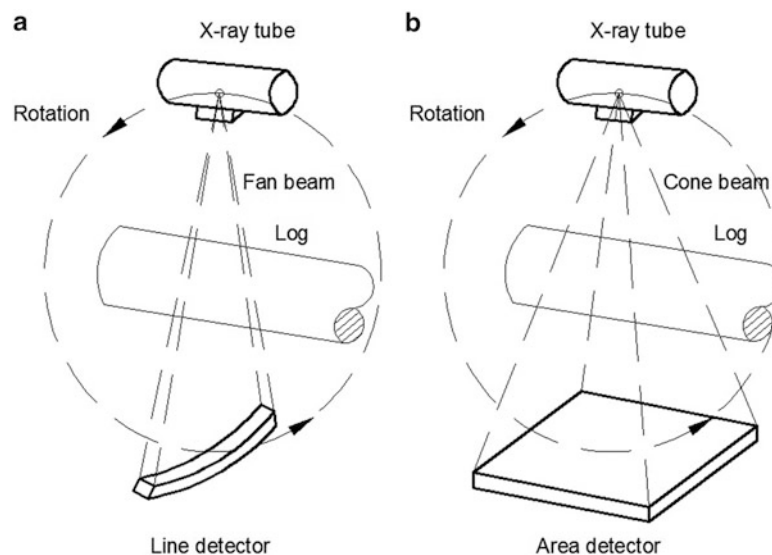


Fig. 2.1 CT scanning geometry: (a) third generation single-slice CT, (b) cone-beam CT

2.3 Cone-Beam Log Scanning Concept

Log CT scanning can be a much more tolerant process than medical CT scanning. Logs are inanimate, so they can be moved and maneuvered conveniently. The features of interest (knots, heartwood/sapwood boundary, rot and holes) are fairly large, mostly in the centimeter range, so that they can be identified using measurements with relatively coarse spatial resolution. Very significantly, logs have specific geometry, which is known in advance (“a-priori”). If used effectively, knowledge of that geometry can greatly simplify and stabilize the CT reconstruction. In addition, the circular symmetry of logs allows easy compensations for lateral rigid-body motions of the logs as they pass through the scanner. This is a very important feature because the handling of the rough logs in sawmills cannot be done with the high precision that is required when doing conventional medical style CT scanning. The practical saw-log CT scanning system described here incorporates these features.

Figure 2.2 shows a schematic of the proposed log CT scanner. The scanner uses a cone-beam collimated X-ray source, a lab-made large area X-ray detector and a log transport and rotation mechanism. The log to be inspected is rotated within the cone-shape illumination space between the X-ray source and detector. During log rotation, X-ray images are taken at a series of incremental angles, from which the log cross-sections within the illumination cone are reconstructed. The arrangement in Fig. 2.2 has two key features: the log rotation mechanism and the large area detector. The log rotation mechanism allows the use of a stationary source and detector, which avoids the difficulty of generating high speed, complex and meticulously controlled rotation of X-ray source and detector needed in medical style equipment. This greatly simplifies the required scanner hardware and makes the design much more practical and robust. The large-area detector dramatically increases the scanned volume, and so makes possible the reconstruction of many slices simultaneously, thereby enabling a much higher scanning speed. Mechanical details of the proposed log CT scanner hardware design and construction will be described in Part II of this paper.

2.4 Geometry-Based Log Models

The a-priori information provided by knowledge of the specific geometry of logs allows substantial computation economy through the use of geometry-based CT models. For most medical and industrial CT scanners, the typical CT geometry is the square-grid pattern shown in Fig. 2.3d, which divides each cross-section into fine-meshed squares. The material density at each square, called a “voxel”, is determined (“reconstructed”) from the X-ray measurements. This generalized geometry is chosen because it accommodates internal structures of any geometry and maximizes the resolution to identify fine features of interest.

In contrast, log scanning doesn’t require very high spatial and density resolution. In particular, logs have specific geometrical shapes: in areas away from knots, they are generally circular with axi-symmetric cross-sectional features and,

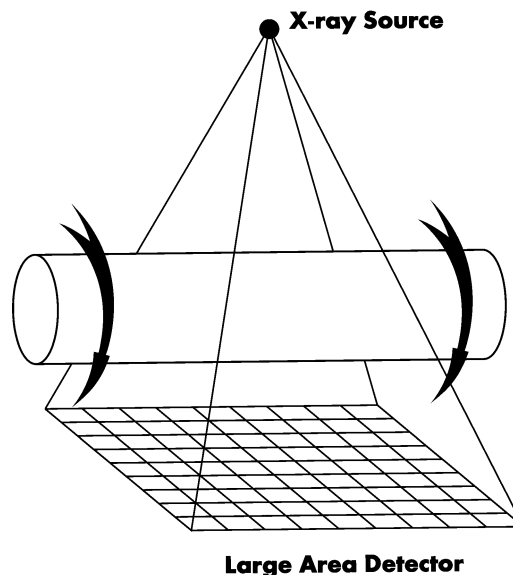


Fig. 2.2 Proposed cone-beam log CT scanning

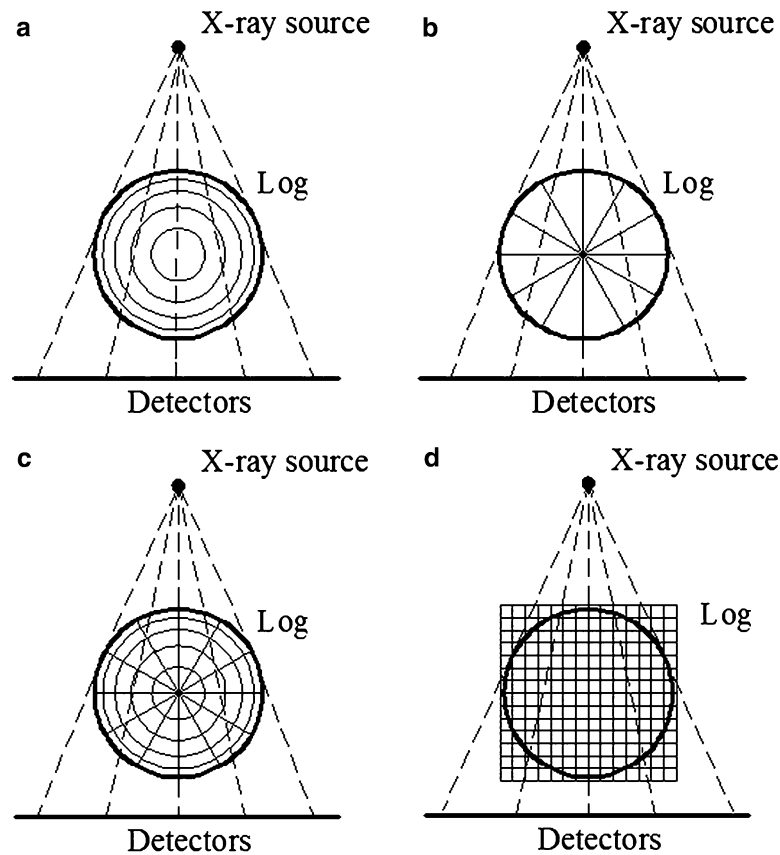


Fig. 2.3 Log geometric CT models: (a) annular model, (b) sector model, (c) combined model, (d) conventional model

where present, the knots start from the center and grow approximately in a sector shape through the perimeter. This a-priori information makes possible the use of much simpler feature-specific log cross-section models instead of the commonly used fine-meshed square pattern. Three coarse-resolution, geometry-based CT log models are proposed here. These three geometric models target different internal features. The first model shown in Fig. 2.3a comprises annular regions. This arrangement is suited for the clear wood regions between knots, where heartwood/sapwood, rings and rot tend to be axisymmetric. The second model shown in Fig. 2.3b comprises sector-shaped regions. This arrangement is suited to the knot regions where the features are sector-shaped. Figure 2.3c shows a combined model, which is suitable when multiple features are present simultaneously. All three log models divide the cross-section into feature-specific regions and tend to guide the resulting cross-sectional reconstructions towards physically realistic solutions. The smaller number of unknown voxels compared with the generic square grid shown in Fig. 2.3d dramatically reduces the quantity of X-ray measurements and the size of the computation.

Two very important features of the geometry-based CT models Fig. 2.3a, b, c are their circular symmetry and their containment of all the voxels within the log boundary. These features enable some pragmatic approximations to be made when processing the measured data to compensate for the lateral rigid-body motions that occur when rough saw-logs are moved within sawmills. This compensation is done by reversing the conventional arrangement where the voxels in Fig. 2.3d are fixed in space and the specimen moves within that space. This arrangement requires very accurate motion of the scanner and specimen. In contrast, the voxels in Fig. 2.3a, b, c are fixed to the log. In the subsequent discussions, it will be shown how this arrangement substantially compensates for lateral rigid-body motions and log non-circularity.

2.5 General CT Computation Description

In each proposed CT model, X-rays fan out from the source, pass through the log and reach the large-area detector. The part of the log within a given X-ray path attenuates the radiation according to the line integral of the density along that path [8]. The relationship between X-ray attenuation and log densities can be expressed using Beer's Law as Eq. 2.1, where I is the

attenuated X-ray intensity, I_0 is the unattenuated intensity, $\rho(x)$ is the log density along the path length at position x , and β is the basis weight coefficient [13, 14].

$$\frac{I}{I_0} = \exp\left(-\int \frac{\rho(x)dx}{\beta}\right) \quad (2.1)$$

Equation 2.1 can be linearized by taking logarithms on both sides:

$$\int \rho(x)dx = -\beta \ln\left(\frac{I}{I_0}\right) \quad (2.2)$$

where the left side of Eq. 2.2 represents the line integral of the material density along the X-ray path. This quantity corresponds to the local basis weight (= density per unit area). After discretization, Eq. 2.2 can be written for the given ray as:

$$\sum g_j \rho_j = d_j \quad (2.3)$$

where g_j is a set of discrete lengths corresponding to material densities ρ_j within a sequence of voxels along the overall ray path. The quantity d_j is the basis weight observed for the given ray according to the measured X-ray attenuation. For the combination of all the rays within the X-ray cone, Eq. 2.3 can be generalized as:

$$\sum G_{ij} \rho_j = d_i \quad (2.4)$$

where G_{ij} is a matrix whose entries represent the path length within ray “ i ” as it passes through voxel “ j ”. This equation can be expressed in vector–matrix format as:

$$[G]\{\rho\} = \{d\} \quad (2.5)$$

where brackets and braces respectively indicate matrix and vector quantities.

2.6 Basis Weight Data Alignment

The cone-beam reconstruction in Eq. 2.5 requires the assembly of data vector $\{d\}$ from the measured basis weight data. If done with some care, this assembly can mostly eliminate the effects of any lateral rigid-body motions of the log within the X-ray beam. Such motions are very damaging in conventional CT scanning and require very accurate control of the scanner or specimen motion. The use of the geometrical CT models in Fig. 2.3a, b, c allow the effects of lateral rigid-body motions to be accommodated mathematically by adjusting the arrangement of the data vector $\{d\}$. The following paragraphs describe the proposed procedure.

2.6.1 Cylindrical Adjustment

For ease of manufacture, the large-area detector used here has a flat detection surface. However, mathematically, it would be more convenient instead to use a curved detection surface in the form of a cylinder whose axis passes through the X-ray source parallel to the longitudinal motion of the log. The measurement points (“pixels”) on the detection surface would then be equally spaced in an angular sense. This gives the detector panel a circular symmetry to complement the circular symmetry of the log model. The log and the detection cylinder have different centers.

Measurements X_i , which are linearly spaced on a flat panel detector, can be mathematically adjusted so that they appear to be angularly spaced θ_i on an equivalent cylindrical detector. This is a fixed geometrical relationship:

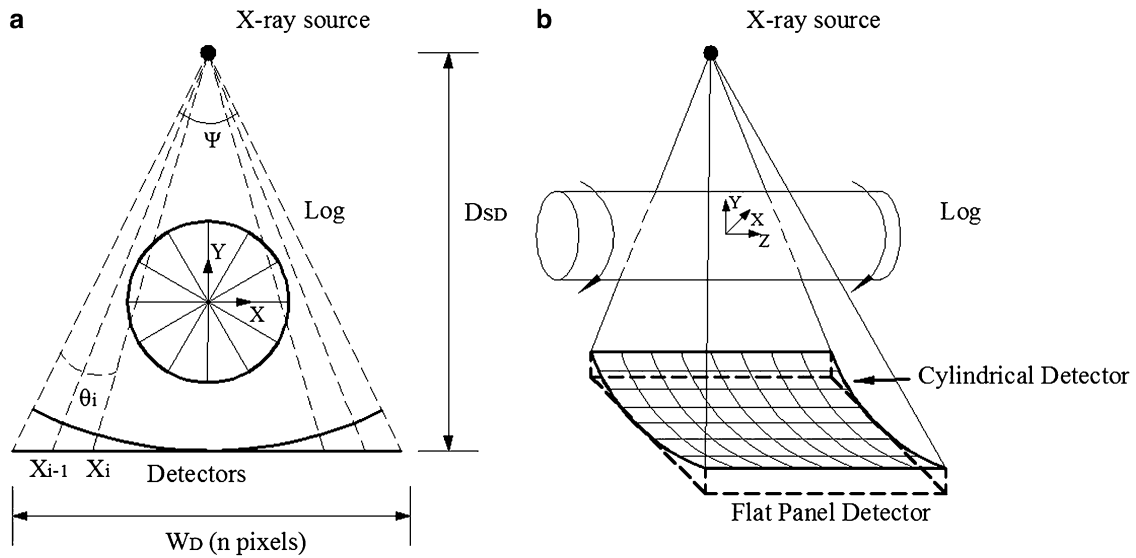


Fig. 2.4 Flat and cylindrical X-ray detectors: (a) axial view, (b) side view

$$X_i = \frac{D_{SD} \tan\left(\theta_i - \frac{\psi}{2}\right)}{\frac{W_D}{n}} + \frac{n}{2} \quad (2.6)$$

where D_{SD} is the distance between source and detector, W_D is the detector width, n is the number of detectors within the detector width, and ψ is the X-ray cone illumination angle subtended by the detector width. The corresponding basis weight $b_a[i]$ at angular pixel “ i ” is evaluated from the integral of the linear basis weights $b(x)$ bounded by the given angular pixel

$$b_a[i] = \frac{\int_{X_{i-1}}^{X_i} b(x) dx}{X_i - X_{i-1}} \quad (2.7)$$

This arrangement is applied column-by-column to all pixels to create the arrangement shown in Fig. 2.4b.

2.6.2 Rigid-Body Motion and Log Ellipticity Correction

The circular symmetry of the cylindrical panel detector shown in Fig. 2.4, together with the log-based CT models in Fig. 2.3, can be exploited to enable corrections of rigid-body motions and non-circular log geometry. For example, consider a small circumferential rigid-body motion of a log relative to the center of the X-ray fan shown in Fig. 2.5a. The effect is to shift the measured radiograph along the arc of the X-ray detector from A-B to A'-B'. The radiograph image seen between A'-B' is the same as would have been seen between A-B had the rigid-body motion not occurred. Thus, a circumferential rigid-body motion can simply be corrected by shifting the radiograph image between pixels A'-B' back to the pixels A-B, which for convenience is assumed to be in the center of the cylindrical panel detector.

Similarly, for radial rigid-body motions such as in Fig. 2.5b, the effect of the motion is to expand the radiograph image A-B to A'-B' (or contract it for an outward radial motion). Thus, a radial rigid-body motion can be corrected by scaling the radiograph image circumferentially between pixels A'-B' to the pixels A-B, and reciprocally scaling the basis weight data so that the same amount of mass is represented within the resulting scaled radiograph image. This scaling concept can be taken a step further to accommodate logs that are slightly elliptical. The arc length A'-B' of the radiograph image in Fig. 2.5c caused by a non-constant log diameter can similarly be scaled to fit the arc length A-B. Thus, this shifting and scaling of the radiograph data around the cylindrical measurement surface can substantially eliminate the effects of rigid-body motions

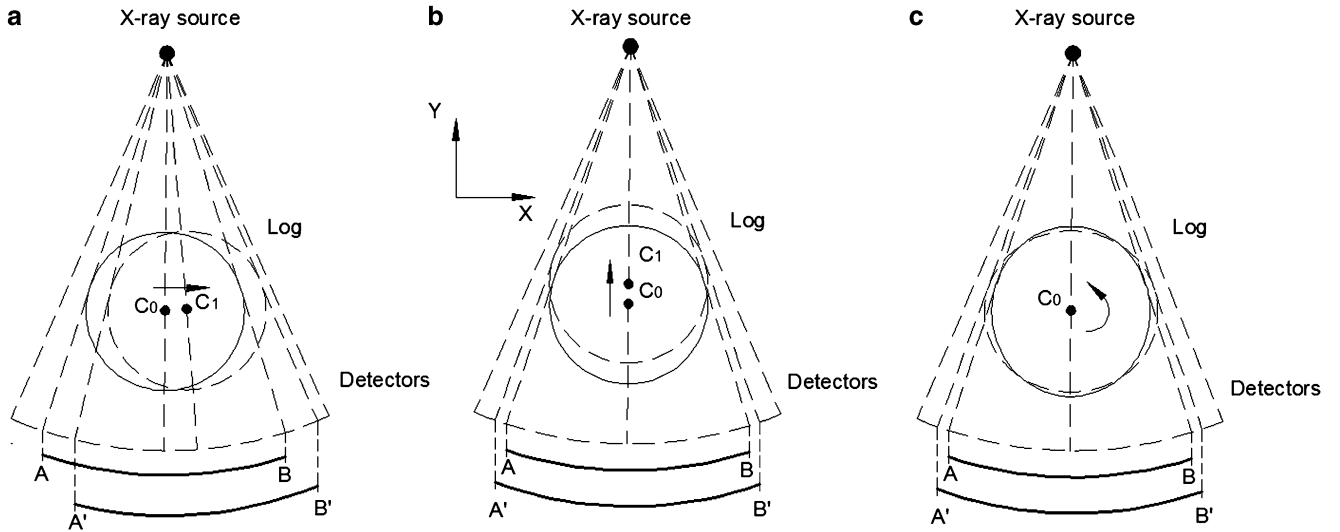


Fig. 2.5 Measurement re-centering and uniform scaling: (a) circumferential motion, (b) radial motion, (c) ellipticity effect

and log non-circularity. The adjustment is not perfect because of small angle changes within the X-ray fan, but for modest rigid-body motions and log diameter variations, the process is quite effective.

2.6.3 Geometry Normalization

The data shifting and scaling shown in Fig. 2.5 opens the opportunity to pursue to concept of arranging Eq. 2.5 in a standardized form, where matrix $[G]$ is the same for all sizes and positions of logs. The idea is to shift the radiograph data so that the log appears as if it were in the center of the field, and to scale the data so that the log appears as if it had a “standard” diameter. The actual diameter of the log ultimately needs to be used to scale the material densities computed for the “standard” log to those of the actual log. Examination of the path lengths shown in Fig. 2.3 shows that the diameter of the log relative to the X-ray source to detector distance does have some influence on relative path lengths, beyond just a simple multiplier based on log diameter. However, for the small ray angles that occur when the X-ray source to detector distance is much greater than the log diameter, this effect is modest.

2.6.4 Log Center and Radius Estimation

The ability to do the rigid-body motion and log ellipticity corrections shown in Fig. 2.5 depends on an ability to identify the log position and diameter within the measured radiograph image. Doing this through image edge detection is unreliable because the image edge information depends on just a small number of local pixels. These may be subject to noise, especially if some local irregularity exists at that point on the log, say due to a branch. A more robust approach is to estimate the log diameter from the entire radiograph image. This can be done by assuming that the log is circular and of uniform density (physically not exactly true, but appears to be good computational approximation). The radiograph profile then has a semi-elliptical shape. Its area is proportional to radius \times axial height, while its centroidal height is proportional to axial height only. Dividing the area by the centroidal height gives a remarkably robust estimate of the log radius:

$$r = D_{SL} \sin \left(\frac{16}{3\pi^2} \frac{d\psi}{\sum (b^2)} \left(\frac{\sum b}{\sum (b^2)} \right)^2 \right) \quad (2.8)$$

where r is the estimated log radius, D_s is the distance between X-ray source to log center, b are the basis weight data, $d\psi$ is the angular spacing between detectors after cylindrical correction, and i is the angular pixel index. The circumferential position of the centroid of the radiograph gives the log center position:

$$c = \frac{\sum ib}{\sum b} \quad (2.9)$$

2.7 Path Length Computation in Cone-Beam CT

Alignment of the data vector $\{d\}$ provides the right side of Eq. 2.5. The next step towards completing the CT reconstruction is to assemble the path length matrix $[G]$.

2.7.1 Single-Slice Path Length Computation

Figure 2.6 illustrates the single-slice path length computation for the three geometry-based CT models in Fig. 2.3. The sequence of annulus radii in Fig. 2.6a, c is in equal increments of the square root of radius. In this way, the areas of the annuli are equal, thereby approximately equalizing the subsequent computational accuracy of the material densities within the annuli. The matrix $[G]$ in Eq. 2.5 has elements G_{ij} , which represent the path length of ray i as it passes through voxel j . Matrix $[G]$ is sparse, with much fewer than half of the elements having non-zero values. The path lengths within the voxels for the annular and sector models in Fig. 2.6a, b can be computed geometrically. The path lengths in the combined model Fig. 2.7c can be computed by evaluating the path lengths using the sector model in Fig. 2.6b with an “log radius” equal to each of the annulus radii in Fig. 2.6c. The path lengths in Fig. 2.6c are equal to the difference of the path lengths of the inner and outer radius sector model path lengths corresponding to each annulus.

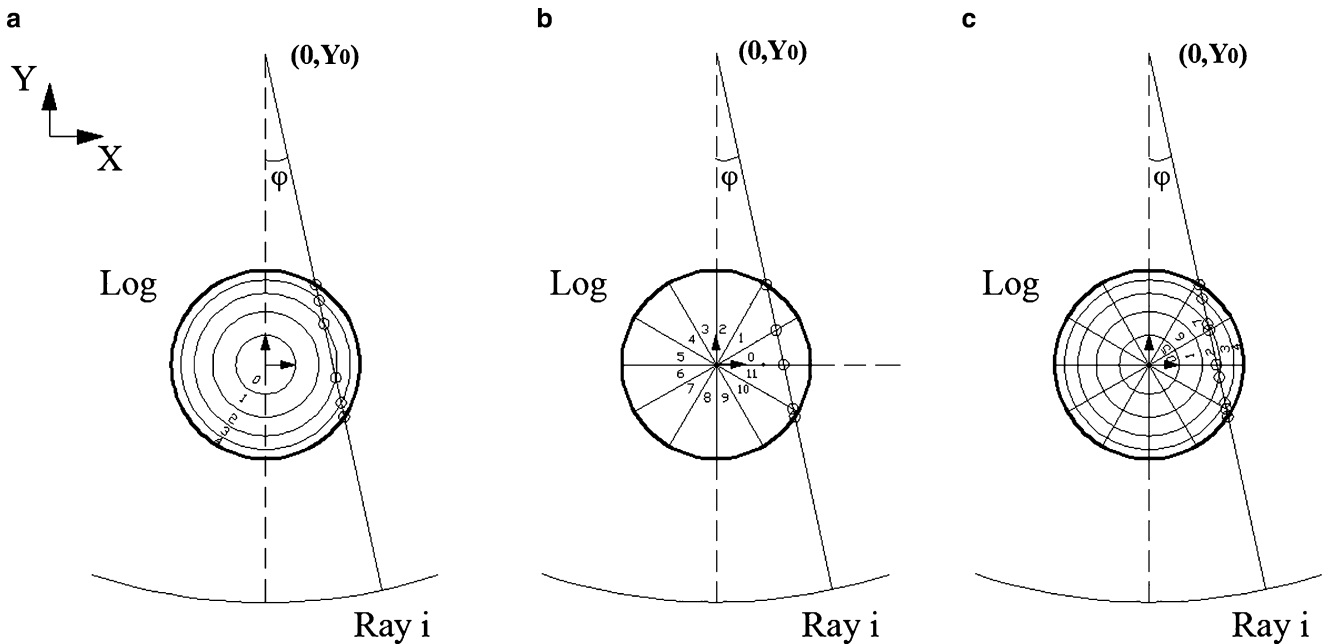


Fig. 2.6 Voxel X-ray path computation: (a) annular model (b) sector model (c) combined model

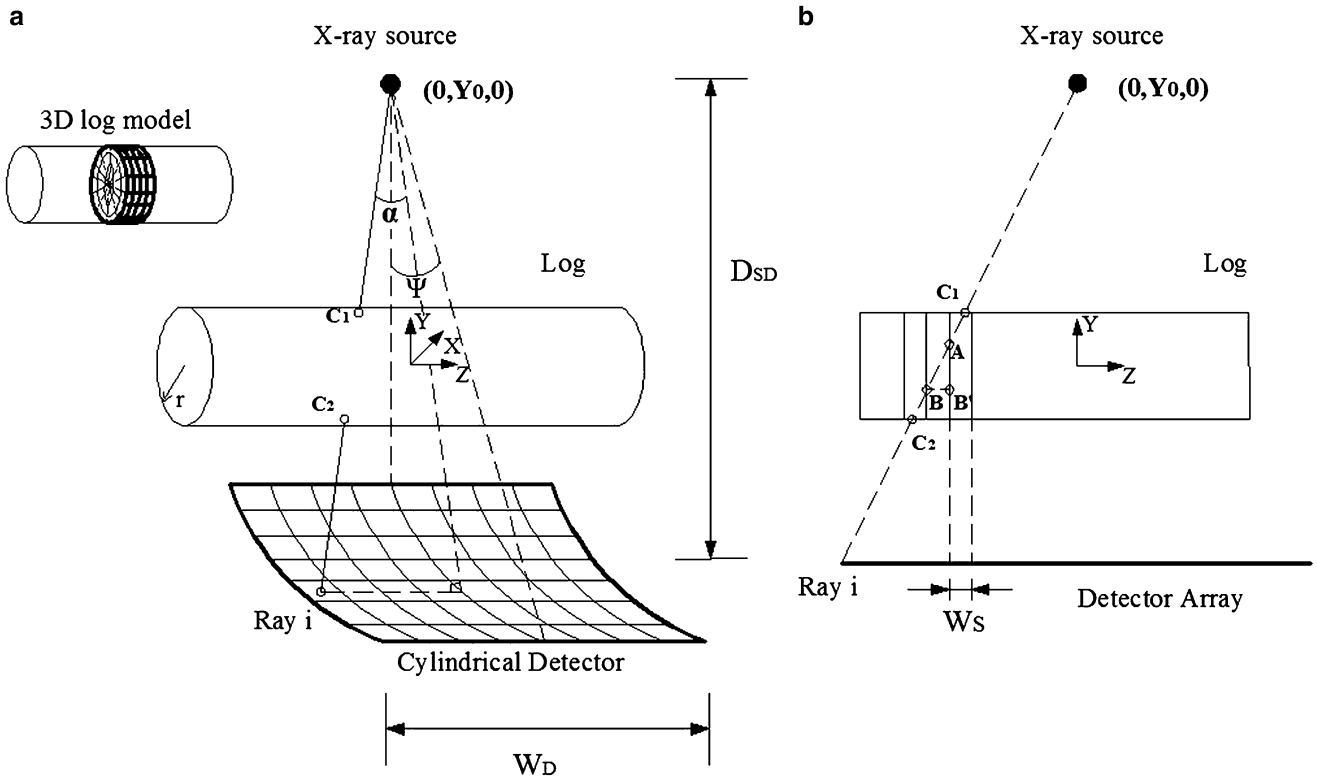


Fig. 2.7 X-ray paths through the log specimen: (a) perspective view, (b) side view

2.7.2 Multiple-Slice Path Length Computation

The geometry shown in Fig. 2.6 is appropriate for CT measurements within single cross-sectional slices. This single-slice approach is used where X-ray measurements are made use line detectors such as in Fig. 2.1a, and is quite common in medical applications. Here, to make fuller use of the X-ray data, the cone-beam arrangement shown in Fig. 2.1b is used. This arrangement creates no conceptual change to the measurement, and the CT reconstruction Eq. 2.5 still applies. However, the path length matrix becomes considerably larger and more computationally intensive to evaluate.

Figure 2.7 illustrates the geometry of the cone-beam scanning configuration. This cone-beam arrangement is very challenging compared with the single-slice arrangement in Fig. 2.6 because the paths of the inclined rays in Fig. 2.7b pass through multiple adjacent slices. This circumstance couples the calculations between adjacent slices so that they no longer can be reconstructed separately; the voxel densities in all slices must be computed simultaneously. This feature greatly multiplies the size of the matrix Eq. 2.5, but fortunately, with the coarse-resolution models used here, the resulting size is still tractable. The resulting path length matrix is very sparse, and if care is taken to accommodate this feature, significant computational economy can be achieved.

Path length computation for cone-beam model in Fig. 2.7 follows the same concept as the computation for the combined model in Fig. 2.6c. First, the inclination of each ray on the cylindrical detector surface is determined geometrically. Then, the radii at which the ray passes through each slice are determined. The path lengths for combined models of the type in Fig. 2.6 are computed, with the path lengths within each slice evaluated by subtraction of the path lengths in adjacent slices.

Computation of the path length matrix $[G]$ is a computationally intensive process, taking some minutes to complete. However, the process is much simplified by the data alignment process described above because the aligned data are placed symmetrically within the scanner geometry, so only one quarter of the path lengths need be computed, and the others assigned by reflections across the two symmetry axes. In addition, for a practical scanner arrangement where the cone-beam angles are small, the data normalization previously described to form a “standard” log geometry means that the path length matrix need be computed only once, stored, and recalled for use with all subsequent logs. Thus, the computation time of matrix $[G]$ is not an issue for real-time use.

2.8 Density Inverse Computation

The path length computations and X-ray data normalization described above provide $[G]$ and $\{d\}$ in Eq. 2.5 for a single X-ray measurement, a “projection”. In general, $[G]$ is not square. For a single projection using the single slice arrangement in Fig. 2.6, the number of rows equals the number of pixel measurements within the cylindrical detector, and the number of columns equals the number of voxels within the geometrical model used. For multiple projections, the number of rows is correspondingly multiplied, but the number of columns remains the same. Substantial computational economy can be achieved by arranging the number of projections to correspond to the number of sectors used in Fig. 2.6b, c. That way the path alignments relative to the sector pattern are the same for each projection, just rotated by an integer number of sectors. Thus, the new rows required for the path length matrix $[G]$ can be simply added by rotating the columns from the path lengths computed for the first projection.

For the multiple-slice arrangement shown in Fig. 2.7, path length matrix $[G]$ expands greatly to have a number of columns equal to the total number of voxels in all slices within the log. The number of rows is much larger, equalling the number of pixels within each panel detector image times the number of images used for the measurement. In conventional CT practice, the resulting massive size of $[G]$ requires the use of inverse solution methods such as filtered back-projection. Here, the number of voxels is much smaller because of their feature-specific shapes, and so a direct solution of Eq. 2.5 is feasible. The resulting highly over-determined Eq. 2.5 can be solved in a least-squares sense as:

$$[G]^T[G]\{\rho\} = [G]^T\{d\} \quad (2.10)$$

where matrix $[G]^T[G]$ is square with row/column size equal to the number of voxels, which is a moderate size number compared with the number of pixels used in all the projections. If care is taken with the sequence of doing the operations $[G]^T[G]$, it is possible to form the result without explicitly storing the very large matrix $[G]$. Actually, it is necessary only to store the part of the matrix for a single projection and then to rotate the columns as required to correspond to the alignment of the subsequent projections. Matrix $[G]^T[G]$ is symmetric positive-definite and can be solved efficiently using a Cholesky solver [15].

2.9 Discussion

The use of the feature-specific geometrical models in Fig. 2.3 for CT reconstructions substantially and fundamentally influences the associated CT inversion. On one level, the geometric resemblance of the models to the physical features of the logs being scanned has the effect of guiding the inversion towards physically realistic results. This important feature will be seen in the experimental results shown in Part II of this paper. On a further and deeper level, the proposed approach is very unusual in that the CT inversion is referenced to the log and not to a fixed volume in space, as is done in conventional practice. By analogy to finite element modeling, the proposed approach may be described as “Eulerian”, while the conventional practice is “Lagrangian”. One consequence is that the thrust of the CT calculation is focused entirely on the volume of the scanned log; the three feature-specific models in Fig. 3a, b, c do not contain any of the external voxels seen in Fig. 2.3d. A further consequence is that the CT inversion becomes fairly tolerant to small rigid-body motions of the log relative to the X-ray source and detector. The measured data from the X-ray detector can be shifted and scaled to transform the data to a “standardized” configuration, for which the path length matrix $[G]$ is predetermined. This scaling is not perfect because change in log diameter changes the angles within the scanned volume in a slightly non-linear way. However, for the small angles used here (16° X-ray cone angle), the non-linearities are modest. This feature will have to be tested in practice. At worst, predetermined path length matrices $[G]$ will be needed for a compact sequence of different log diameters to accommodate large changes in log size.

The shifting and scaling of the X-ray data to a standardized format enable the CT inversion to be done in a normalized form. This is a great mathematical convenience. However, the results ultimately need to be referenced to the actual log diameter to identify true physical dimensions. The X-ray measurements do provide an indication of the physical dimensions, but the presence of radial rigid-body motions within a cone-beam geometry change the size of the scanned log image and therefore impedes precise identification of dimensions. Multiple X-ray systems from different angles could be used to resolve the ambiguity, but this would be a rather complex and expensive solution. A more practical approach is to use an added optical scanner to identify log size and position. Such optical scanners are widely used in the wood industry and are rugged and relatively inexpensive. Ultimately, the tolerance of small rigid-body motions is significant because it greatly

simplifies the required mechanical arrangement for the needed X-ray scanner. In conventional CT practice, the measured object stays stationary while the X-ray scanner system rotates around it. For medical applications, this approach is essential because the objective of the measurement is a human patient, which cannot be conveniently rotated! But even in non-medical applications, the same approach is used because the X-ray system can be rotated with much higher precision than then measured object. This is certainly the case with logs in a sawmill. The logs are large and rough and cannot be moved with great precision. The tolerance of rigid-body motions by the proposed CT arrangement makes acceptable the imprecision of the log motions available in a sawmill and very greatly reduces the complexity, fragility and cost of the required X-ray scanner.

2.10 Conclusion

This paper, the first of a two-part series, introduces the novel cone-beam log scanning concept, the coarse-resolution, geometry-based log models, the associated algorithms and computation procedures. This proposed Eulerian approach has several advantages over conventional medical style CT scanning:

1. It reduces scanner mechanical complexity by using a stationary X-ray source and detector while rotating the measured object.
2. It uses feature-specific voxel geometries to guide and stabilize the CT reconstruction.
3. It is tolerant of rigid-body-motions and log ellipticity.
4. It reduces the number of features that need to be determined, thus reducing the scale of computation and effort for data processing.
5. It used all data available within the X-ray cone beam, thus increasing the data content and stability of the CT inversion.

The practical demonstration of proposed log CT system will be introduced as the second part of this serial paper.

Acknowledgments The authors gratefully thank the Natural Science and Engineering Research Council of Canada (NSERC) for their financial support of this project through the ForValueNet network, the Centre for Hip Health and Mobility, Vancouver General Hospital, for use of facilities, and the Institute for Computing, Information and Cognitive Systems (ICICS).

References

1. Oja J, Grundberg S, Fredriksson J, Berg P (2004) Automatic grading of sawlogs: a comparison between X-ray scanning optical three-dimensional scanning and combinations of both methods. *Scand J Forest Res* 19:89–95
2. Pietkäininen M (1996) Detection of knots in logs using X-ray imaging. Dissertation, Technical Research Centre of Finland, Espoo, VTT Publications 266
3. Usenius A (2003). Optimization of sawing operation based on internal characterization of the logs. In: Proceedings ScanTech 2003, Wood Machining Institute, Seattle, pp 11–18
4. Rinnhofer A, Petutschnig A, Andreu JP (2003) Internal log scanning for optimizing breakdown. *Comput Electron Agric* 41:7–21
5. Chiorescu S, Grönlund A (2000) Validation of a CT-based simulator against a sawmill yield. *Forest Prod J* 50:69–76
6. Oja J, Wallbacks L, Grundberg S, Hagerdal E, Grönlund A (2003) Automatic grading of Scots pine (*Pinus sylvestris* L.) sawlogs using an industrial X-ray log scanner. *Comput Electron Agric* 41:63–75
7. Seger MM, Danielson PE (2003) Scanning of logs with linear cone-beam tomography. *Comput Electron Agric* 41:45–62
8. Lindgren LO (1991) Medical CAT-scanning X-ray absorption coefficients, CT-number and their relation to wood density. *Wood Sci Technol* 25:341–349
9. Som S, Wells P, Davis J (1992) Automated feature extraction of wood from tomographic images. In: Proceeding of international conference on automation, robotics and computer vision, Singapore, 15–18 Sept 1992, pp CV-14.4.1–CV-14.4.5
10. Krahenbuhl A, Kerautret B, Longuetaud F (2011) Knots detection in X-ray CT images of wood. *Scand J Forest Res* 12:80–90
11. Longuetaud F, Mothe F, Kerautret B (2012) Automatic knots detection and measurements from X-ray CT images of wood: a review and validation of an improved algorithm on softwood samples. *Comput Electron Agric* 85:77–89
12. Wang G, Yu H (2008) An outlook on X-ray CT research and development. *Med Phys* 35(3):1051–1064
13. Kak AC, Slaney M (1987) Principles of computerized tomography. IEEE Press, New York
14. ASTM (1993) Standard guide for computed tomography (CT) imaging. American Society for Testing and Materials, West Conshohocken. ASTM Standard E-1441-11, 33pp
15. Galassi M et al (2009) GNU scientific library reference manual, 3rd edn. <http://www.gnu.org/software/gsl/>. Accessed 25 June 2011

Chapter 3

Coarse-Resolution Cone-Beam Scanning of Logs Using Eulerian CT Reconstruction. Part II: Hardware Design and Demonstration

Yuntao An and Gary S. Schajer

Abstract Computed Tomography (CT) provides an attractive opportunity in the wood industry for quality assessment of logs entering a sawmill to enable the material to be appropriately processed into maximum value products. A CT scanner suitable for this task must be simple and economical, considerably more so than the complex and costly scanners typically used for medical applications. Part I of this paper presented the design concept and the theory of operation of a specially designed coarse-resolution cone-beam CT scanner. This second part focuses on the CT system hardware design, construction and performance. Key components are a large-format custom-designed X-ray detector panel, a log spiral-motion mechanism and a real-time data acquisition system. These are described in detail. The coarse-resolution reconstruction results for an example log using feature-specific models and algorithms introduced in Part I of this paper are demonstrated and they compare well with CT reconstructions for same log using the same measurement with conventional filter back projection algorithm. The good comparison gives confidence in the usefulness and applicability of the proposed CT scanner design for industrial use in sawmills for log quality assessment.

Keywords Log CT scanner • Large area X-ray detector • Cone-beam CT • Coarse-resolution CT • Log feature identification

3.1 Introduction

Wood is a natural material and has highly variable structure. To process it efficiently in sawmills, it is essential to be able to identify internal features such as knots, heartwood/sapwood boundary, rot and holes, so that appropriate cutting decisions can be made. The highest value products can then be made from the given raw material. Some interesting log scanning systems have appeared on the market, among which are some that use X-ray imaging by one or more line detectors viewing the log from various angles [1, 2]. X-ray images only provide 2-D information, so the features in the third direction are not well identified [3]. Although using multiple X-ray sources and detectors at different angles allows better perception of internal features, this approach greatly increases the scanner system cost. In addition, the limited number of measurements provides only limited capacity to identify log internal features. Several researchers have indicated that successful application of CT log scanning in sawmill can greatly improve productivity and profit [4, 5]. However directly applying existing CT technology to saw-log scanning is not an easy task to accomplish [6].

Commercial CT systems modeled on medical designs and don't fit well with sawmill operation environment. They are very expensive, beyond the budget of most sawmills. They are also mechanically complex and delicate, require high-precision motions of sensors and specimen and thus won't tolerate the severe working environment in a sawmill. They also require massive data collection, processing and intensive CT reconstruction computations, thus are challenged by the extreme scanning speed required for real-time operation.

In Part I of this paper [7], the use of X-ray CT scanning is explored and the particular challenges of using this technique in sawmills are identified. These challenges include the need of the CT system to be mechanically robust and straightforward, to be tolerant of material handling inaccuracies, and to be of modest cost. All these needs do not align well with the characteristics of conventional CT scanners designed for medical use, which tend to be complex, highly precise and very costly devices.

Y. An • G.S. Schajer (✉)

Department of Mechanical Engineering, University of British Columbia, Vancouver, BC, Canada

e-mail: schajer@mech.ubc.ca